

Contactless Rheology of Soft Gels Over a Broad Frequency Range

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(Received 10 February 2022; revised 12 April 2022; accepted 5 May 2022; published 23 June 2022)

We report contactless measurements of the viscoelastic rheological properties of soft gels. The experiments are performed using a colloidal-probe atomic force microscope in a liquid environment and in dynamic mode. The mechanical response is measured as a function of the liquid gap thickness for different oscillation frequencies. Our measurements reveal an elastohydrodynamic coupling between the flow induced by the probe oscillation and the viscoelastic deformation of the gels. The data are quantitatively described by a viscoelastic lubrication model. The frequency-dependent storage and loss moduli of the polydimethylsiloxane gels are extracted from fits of the data to the model and are in good agreement with the Chasset-Thirion law. Our results demonstrate that contactless colloidal-probe methods are powerful tools that can be used for probing soft interfaces finely over a wide range of frequencies.

DOI: 10.1103/PhysRevApplied.17.064045

I. INTRODUCTION

Mechanical properties of thin, soft solids are of great interest for many emerging applications including surface coatings [1–3], photonics [4,5], microelectronics [6,7], and biosensors [8–10]. Methods like tensile or bending tests and direct indentation are commonly used for the characterization of soft materials [11–14]. The atomic force microscope (AFM) is a widely used device for surface topography and is also used for mechanical measurements with a nanometric resolution. AFM probes, with known shapes, can be used as nanoindenters to measure the mechanical response of thin films under applied normal compression [15–23]. With such a device, one can perform nanomechanical characterization of materials with elastic moduli in the range from kPa to GPa.

However, there are limitations in the measurement of elastic moduli for soft samples using the direct contact method due to the adhesion between sample and probe, which is inevitable at the nanoscale [23–26]. It is difficult not only to accurately measure adhesion, but also to measure the area of an adhesive contact [12,27,28]. Sample damage and probe contamination may also occur during measurements of soft samples [29].

Contactless measurements are an alternative to direct-contact methods. More precisely, elastohydrodynamic

(EHD) interactions between a probing sphere and a soft sample in a liquid environment provide a precise method to assess mechanical properties [30,31]. At small liquid-gap thickness between the sphere and the soft substrate, the vibration of the former generates a hydrodynamic stress field that might be large enough to deform the substrate, which in turn perturbs the flow leading to an EHD coupling. Based on this coupling, some tools have been developed to probe the mechanical properties of soft interfaces without contact. Using surface force apparatus, the mechanical properties of substrates of various rigidities, ranging from soft elastomeric samples to hard glasses, have been measured at low frequencies (less than 100 Hz) [32,33]. Guan *et al.* have used a vibrating nanoneedle glued onto an AFM cantilever to probe the viscoelastic properties of polydimethylsiloxane (PDMS) samples [34] and living cells [35]. However, in the AFM measurements of Guan *et al.* [34,35], the experimental setup restricts the probed frequency to a single value given by the resonance frequency of the cantilever and usually in the range of tens of kilohertz. With the setup proposed by Guan *et al.*, the viscoelastic properties of a sample can only be measured at a single high frequency. Nanomechanical resonators were also used to directly probe the stiffnesses of various samples [36,37], but the measurements were limited to the resonance frequencies of the resonators.

Usually, the frequency-dependent rheology of soft solids is probed at the macroscale with conventional shear

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rheometers, where the sample is fixed between two rotating parallel plates [38–41]. Imposing a rotational displacement to one plate and measuring the applied torque on the other plate allows one to extract the complex shear modulus of the system, $\mu(\omega) = \mu'(\omega) + j\mu''(\omega)$, where ω is the angular frequency, $\mu'(\omega)$ is the storage shear modulus, $\mu''(\omega)$ is the loss shear modulus [42–44], and j is the imaginary number such that $j^2 = -1$. This method requires perfect contact between the sample and the plates of the rheometer.

In contrast, the dynamic colloidal AFM method [45] is a good candidate for probing the substrate rheology at the nanoscale for different frequencies and without such a constraint. It offers the key advantage of a large frequency range using the same colloidal probe, from 20 Hz up to a few kilohertz.

In this paper, we present measurements of the mechanical response of immersed, thin, soft polymeric gels using an AFM colloidal probe method as a function of the separation distance between the probe and the soft substrate. From the mechanical impedance, we obtain simultaneously the loss and storage moduli of the soft solid. The measured viscoelastic properties of thin PDMS samples at different frequencies are in good agreement with the Chasset-Thirion model. Our study demonstrates that the contactless colloidal probe is a powerful tool to characterize soft interfaces in a noninvasive manner and over a large range of frequencies.

II. EXPERIMENT

The studied samples are PDMS substrates prepared as follows. First, uncrosslinked PDMS (Sylgard 184, Dow Corning) and a curing agent are mixed with a weight ratio of 70:1. Following a degassing process in vacuum, a droplet of the mixture is spin-coated on a cover slide with a size of 24 × 24 mm for a minute, to get a sample with thickness $e = 26 \pm 3 \mu\text{m}$. The sample is then annealed in an oven at 50 °C and for 24 h, to promote an efficient cross-linking. The employed viscous liquid is dodecane with a viscosity of $\eta = 1.34 \text{ mPa s}$. After swelling, the measured thickness of the sample is $125 \pm 5 \mu\text{m}$.

The experiments are performed using an AFM (Bioscope, Bruker) equipped with a liquid cell (DTFML-DDHE) that allows one to perform measurements in a liquid medium and that provides a clean cantilever excitation spectrum (free from spurious peaks). A schematic of the experimental setup is shown in Fig. 1. A borosilicate sphere with a radius $R = 55 \pm 1 \mu\text{m}$ is glued at the end of a silicon nitride cantilever (SNL-10, Bruker), and the stiffness of the cantilever is determined by the drainage method to be $k_c = 0.25 \pm 0.02 \text{ N/m}$ [46]. The resonance frequency and bulk quality factor are obtained from thermal spectral density as $\omega_0/(2\pi) = 1375 \pm 5 \text{ Hz}$ and $Q = 2.8 \pm 0.1$, respectively. The sample is fixed on a piezo stage (NanoT series, Mad City Labs), which is used to

control the distance between the sphere and the sample by imposing a displacement to the substrate at low velocity (less than 160 nm/s). The cantilever is excited by the base oscillation $A_b e^{j\omega t}$ [45,47], where A_b and ω are the amplitude and angular frequency of the base oscillation, respectively, and where t denotes time. The amplitude A and phase φ of the oscillation of the cantilever are measured by a lock-in amplifier (Model 7280, signal recovery) versus the piezo displacement. In addition, the dc component of the cantilever's deflection is also recorded to determine the distance between the sphere and the sample surface.

The system is modeled as a forced damped harmonic oscillator, and the vertical displacement $Z(t)$ of the sphere satisfies [45,47]

$$m^* \ddot{Z} + \gamma_b \dot{Z} + k_c Z = F_d + F_h, \quad (1)$$

where m^* is the effective mass of the probe, γ_b is the damping coefficient in the bulk liquid, k_c is the cantilever's stiffness, F_d is the driving force acting on the cantilever, and F_h is the EHD force induced by the flow between the sphere and the deformable sample. Using the ansatz $Z(t) = A e^{j(\omega t + \varphi)} + A_b e^{j\omega t}$, the complex mechanical impedance $G = G' + jG''$ can be defined and determined by [45,48]

$$G = -\frac{F_h}{Z} = -k_c \left[1 - \left(\frac{\omega}{\omega_0} \right)^2 + j \frac{\omega}{\omega_0 Q} \right] \frac{A e^{j\varphi} - A_\infty e^{j\varphi_\infty}}{A e^{j\varphi} + A_b}, \quad (2)$$

where A_∞ and φ_∞ are respectively the amplitude and phase measured far from the surface, i.e., where the EHD force vanishes.

Figure 2 shows typical acquired data. Figure 2(a) presents the amplitude and phase (A_∞, φ_∞) versus the oscillation frequency, far from the surface ($D > 400 \mu\text{m}$), where the EHD interaction between the substrate and the sphere can be neglected. We observe the presence of spurious peaks (related to the mechanical resonances of the AFM) superimposed on the resonance excitation spectrum of the cantilever. The experiments are performed with frequency values avoiding such spurious peaks. Figure 2(b) presents the measured amplitude A (red dots) and phase φ (green dots) as functions of the average separation distance D for an oscillation frequency $\omega/(2\pi) = 500 \text{ Hz}$. Additional measurements (blue dots and black dots, respectively) on a hard surface (silicon) are used to obtain the base oscillation A_b . Indeed, the silicon surface is hard enough for us to neglect the sample deformation. Thus, in that case, the interaction is purely viscous and can be modeled by a Reynolds force ($G' = 0$, $G'' = 6\pi\eta R^2\omega/D$) [45,49]. Very close to the silicon surface, the imaginary component G'' of the impedance tends to infinity ($D \rightarrow 0$, $G'' \rightarrow \infty$), and the amplitude [blue dots in

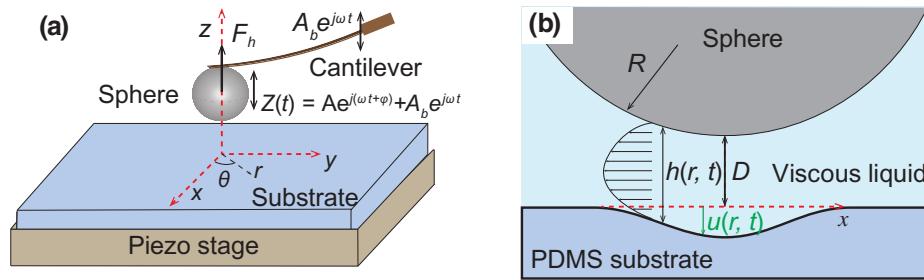


FIG. 1. (a) Schematic of the experimental setup. A borosilicate sphere is glued on the edge of an AFM cantilever, which is vibrated vertically near a soft PDMS substrate within a viscous liquid (dodecane, $\eta = 1.34 \text{ mPa s}$) by the base oscillation $A_b e^{j\omega t}$ with an amplitude A_b and an angular frequency ω . (b) The motion of the sphere gives rise to an axisymmetric liquid-gap thickness profile $h(r, t)$ depending on the radial position r and time t , and related to the sample deformation $u(r, t)$, the average distance D , and the sphere vibration $Z(t)$.

Fig. 2(b)] of the cantilever oscillation tends to the value of the base oscillation ($A \rightarrow A_b$) [45]. In practice, in Fig. 2(b), the amplitude of base oscillation is determined as $A_b = 3.1 \text{ nm}$ at $\omega/(2\pi) = 500 \text{ Hz}$. Once this parameter is measured, the complex mechanical impedance G can be computed using Eq. (2).

III. ELASTOHYDRODYNAMIC MODEL

We focus on the regime of small distance, i.e., $D \ll R$, but larger than a size of several molecules (tens of nanometers), which allows us to neglect any confinement effect on the viscosity. In the framework of the lubrication approximation, and no-slip boundary condition on the sphere and substrate surfaces, the liquid thickness expression $h(r, t)$ is given by the Reynolds equation [49]:

$$\frac{\partial h(r, t)}{\partial t} = \frac{1}{12\eta r} \frac{\partial}{\partial r} \left(rh(r, t)^3 \frac{\partial}{\partial r} p_e(r, t) \right), \quad (3)$$

where $p_e(r, t)$ is the excess hydrodynamic pressure with respect to the atmospheric pressure, with r and t the radial coordinate and time, respectively. The liquid-gap thickness [see Fig. 1(b)] is related to the substrate's deformation $u(r, t)$ and the sphere's vibration $Z(t)$, through [49]

$$h(r, t) = D + \frac{r^2}{2R} + Z(t) + u(r, t). \quad (4)$$

The amplitude of the base oscillation and the sample deformation are very small compared with the average gap distance D , and thus Eq. (3) can be linearized to get

$$\frac{dZ(t)}{dt} + \frac{\partial u(r, t)}{\partial t} = \frac{1}{12\eta r} \frac{\partial}{\partial r} \left(r \left(D + \frac{r^2}{2R} \right)^3 \frac{\partial}{\partial r} p_e(r, t) \right). \quad (5)$$

The cantilever's motion, the substrate's deformation, and the excess pressure are assumed to have harmonic time dependences, in the form $Z(t) = Z_0 e^{j\omega t}$, $u(r, t) = U(r) e^{j\omega t}$, and $p_e(r, t) = P(r) e^{j\omega t}$, respectively. Equation (5)

then becomes

$$12j\eta r\omega(Z_0 + U(r)) = \frac{d}{dr} \left(r \left(D + \frac{r^2}{2R} \right)^3 \frac{d}{dr} P(r) \right). \quad (6)$$

Integrating Eq. (6) once, we get [49]

$$\frac{d}{dr} P(r) = \frac{6j\eta\omega r Z_0}{(D+(r^2/2R))^3} + \frac{12j\eta\omega}{r(D+(r^2/2R))^3} \int_0^r r' U(r') dr'. \quad (7)$$

The substrate is assumed to be a semi-infinite viscoelastic medium with Poisson ratio ν and complex Young's modulus $E(\omega)$, including the frequency-dependent storage modulus $E'(\omega)$ (real part) and loss modulus $E''(\omega)$ (imaginary part). Therefore, the deformation is related to the pressure field through [34,49,50]

$$\hat{U}(k) = \frac{2\hat{P}(k)}{E^*(\omega) k}, \quad (8)$$

where $E^*(\omega) = E(\omega)/(1 - \nu^2)$ is the reduced Young's modulus, and where $\hat{U}(k)$ and $\hat{P}(k)$ are the zeroth-order Hankel transforms of the deformation $U(r)$ and the pressure $P(r)$, respectively, defined as $\hat{U}(k) = \int_0^{+\infty} J_0(kr) U(r) r dr$ and $\hat{P}(k) = \int_0^{+\infty} J_0(kr) P(r) r dr$, with J_0 the zeroth-order Bessel function of the first kind. For incompressible materials, $\nu = 0.5$ and thus $E^* = 4/3(E)$. In such a case, and inverting the transformation, we get

$$U(r) = \int_0^\infty J_0(kr) \hat{U}(k) k dk = \int_0^\infty J_0(kr) \frac{3\hat{P}(k)}{2E(\omega)} dk. \quad (9)$$

Injecting Eq. (9) into Eq. (7) and using the relation between the zeroth-order and first-order (J_1) Bessel functions of the first kind, i.e., $\int_0^r J_0(kr') r' dr' = r J_1(kr)/k$, we get

$$\frac{d}{dr} P(r) = \frac{6j\eta\omega r Z_0}{(D+(r^2/2R))^3} + \frac{18j\eta\omega}{(D+(r^2/2R))^3 E(\omega)} \int_0^\infty \frac{J_1(kr) \hat{P}(k)}{k} dk. \quad (10)$$

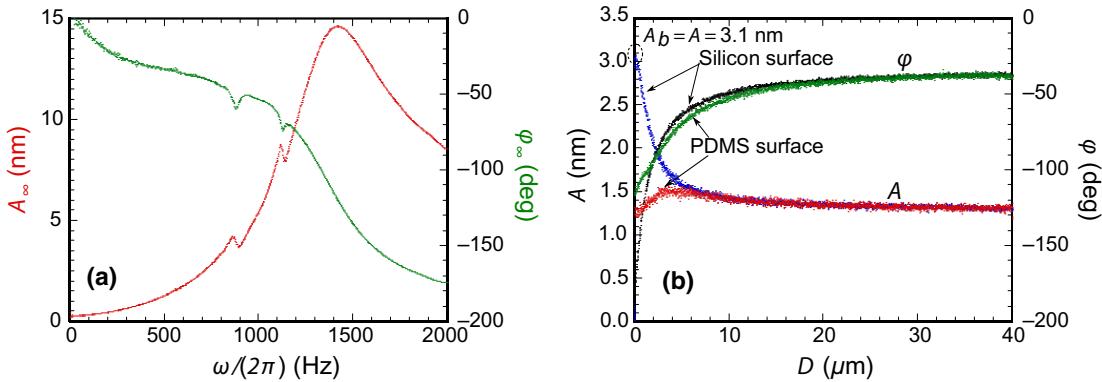


FIG. 2. (a) Amplitude and phase of the cantilever's oscillation versus oscillation frequency measured far from the surface ($D > 400 \mu\text{m}$). (b) Amplitude and phase measured as functions of the average separation distance between the sphere and either a soft PDMS substrate or a rigid silicon substrate. When the sphere is very close to the silicon surface, the amplitude A is equal to the base amplitude $A_b = 3.1 \text{ nm}$.

We further introduce the following dimensionless variables:

$$x = \frac{r}{\sqrt{2RD}}, q = k\sqrt{(2RD)}, p = \frac{PD^2}{\eta RZ_0\omega},$$

$$D_c = 8R\left(\frac{3\eta\omega}{4E'(\omega)}\right)^{2/3}, B(\omega) = \frac{E''(\omega)}{E'(\omega)}.$$

Using these variables, Eq. (10) becomes

$$\frac{d}{dx}P(x) = \frac{12jx}{(1+x^2)^3} + \frac{3j}{(1+x^2)^3}\left(\frac{D_c}{D}\right)^{3/2}\frac{1-jB}{1+B^2}\int_0^{+\infty}\frac{\hat{p}(q)J_1(qx)}{q}dq,$$
(11)

where $\hat{p}(q)$ is the zeroth-order Hankel transform of $p(x)$. A first-order Hankel transform of Eq. (11) leads to a Fredholm integral equation for $\hat{p}(q)$:

$$\hat{p}(q) = \frac{3j}{2}qK_1(q) + 3j\left(\frac{D_c}{D}\right)^{3/2}\frac{1-jB}{1+B^2}\int_0^\infty\hat{p}(q')dq'$$

$$\times\int_0^{+\infty}\frac{J_1(q'x)J_1(qx)}{(1+x^2)^3qq'}xdx,$$
(12)

where K_1 is the first-order modified Bessel function of the second kind. The mechanical impedance is then given by [49]

$$G(D) = -\frac{1}{Z_0}\int_0^{+\infty}2\pi rP(r)dr = -\frac{4\pi\eta R^2\omega}{D}\hat{p}(0),$$
(13)

which can be rescaled as

$$G(D) = \frac{6\pi\eta R^2\omega}{D_c}g_c\left(\frac{D}{D_c}, B\right), g_c\left(\frac{D}{D_c}, B\right) = -\frac{4D_c}{6D}\hat{p}(0).$$
(14)

The dimensionless function $g_c(D/D_c, B)$ can be computed numerically from Eq. (12) [49]. Doing so, we obtain the

mechanical impedance. In particular, at large distance, the mechanical response is dominated by the viscous contribution; hence, we get the asymptotic expression of the impedance:

$$G(D) \simeq \frac{6\pi\eta R^2\omega}{D}\left(j + \frac{9\pi^2}{512}\left(\frac{D_c}{D}\right)^{3/2}\frac{1}{1+B^2}\right).$$
(15)

IV. RESULTS AND DISCUSSION

Figure 3 shows the dimensionless mechanical impedance $GD_c/(6\pi\eta R^2\omega)$ as a function of the dimensionless distance D/D_c , for two frequencies. Figure 3(a) shows the results obtained from the data shown in Fig. 2(b) and using Eq. (2), where the oscillation frequency is $\omega/(2\pi) = 500 \text{ Hz}$. The solid black lines correspond to Eq. (14), where the values of the storage modulus $E' = 3.2 \pm 0.5 \text{ kPa}$ and loss modulus $E'' = 2.0 \pm 0.2 \text{ kPa}$ are the fitting parameters. Figure 3(b) shows the results for the frequency $\omega/(2\pi) = 50 \text{ Hz}$. Similarly, we obtain the values of the storage modulus $E' = 1.6 \pm 0.4 \text{ kPa}$ and loss modulus $E'' = 0.6 \pm 0.2 \text{ kPa}$.

Furthermore, at large distances ($D > D_c$), the viscous component G'' of the mechanical impedance dominates and follows an approximately D^{-1} scaling law [see Eq. (15)], corresponding to the asymptotic expression of the hydrodynamic damping between a sphere and a rigid plane [49]: $G''(D) = 6\pi\eta R^2\omega/D$. Nevertheless, the deformation of the substrate leads to a nonzero elastic component G' of the mechanical impedance, that follows an approximately $D^{-5/2}$ scaling law, in good agreement with the asymptotic prediction of the viscoelastic lubrication model [see Eq. (15)].

At small distances ($D < D_c$), both G' and G'' saturate to constant values that are independent of the average gap thickness. This is due to the fact that the deformation of the soft sample cannot exceed the oscillation amplitude of the sphere, and thus saturates, leading to a saturation

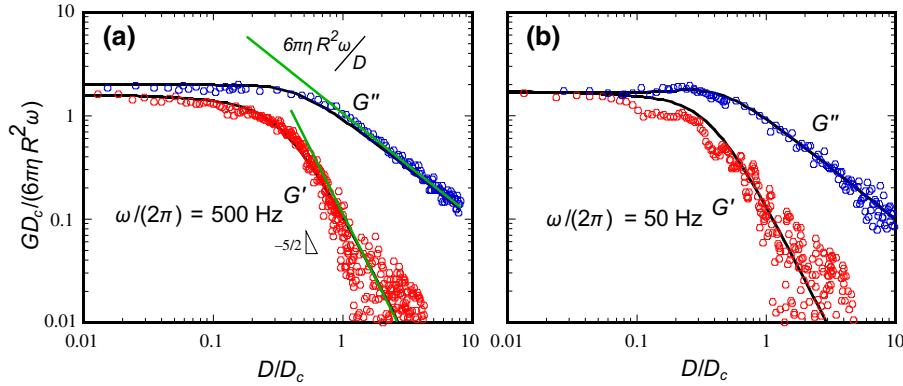


FIG. 3. Normalized real and imaginary parts of the mechanical impedance as functions of normalized distance, for two oscillation frequencies. The solid lines are fitting curves using Eq. (14). (a) Results for an oscillation frequency of 500 Hz, as obtained from the data shown in Fig. 2(b) and Eq. (2). The extracted free parameters are the storage and loss moduli, $E' = 3.2 \pm 0.5$ kPa and $E'' = 2.0 \pm 0.2$ kPa, respectively. (b) Results for an oscillation frequency of 50 Hz. The extracted moduli are $E' = 1.6 \pm 0.4$ kPa and $E'' = 0.6 \pm 0.2$ kPa.

of the excess pressure. In this near-contact regime, the sample deformation accommodates the sphere motion, and the liquid is no longer expelled from the gap. Besides a residual damping that occurs outside the gap due to the viscous flow therein, another part of the damping occurs inside the soft substrate itself due to the loss modulus of the PDMS gel. This latter contribution increases as the oscillation frequency increases and dominates at high frequencies. From the numerical evaluation of Eq. (14), and in the small B case (i.e., $B < 2$, which is valid for the samples employed here), the limiting value of G'' at small distances ($D/D_c \rightarrow 0$) is found to depend on the moduli of the viscoelastic substrate via the relation

$$\left\{ \begin{array}{l} G' \simeq \frac{6\pi\eta R^2\omega}{D_c}(2.01 - 0.77B(\omega)), \\ G'' \simeq \frac{6\pi\eta R^2\omega}{D_c}(1.16 + 1.34B(\omega)). \end{array} \right. \quad (16)$$

Moreover, comparing Figs. 3(a) and 3(b), we observe that for the lower oscillation frequency, the real and imaginary parts of the mechanical impedance merge and saturate to the same value at small distances, whereas for the higher frequency the imaginary part remains larger than the real part at all distances. Furthermore, the imaginary part of the mechanical impedance increases as the frequency increases. The dissipation is thus more pronounced at large frequency.

The extracted storage modulus E' and loss modulus E'' for various oscillation frequencies are shown in Fig. 4. Both moduli increase as the frequency increases. In our experiment, the PDMS sample is a viscoelastic material, and the frequency dependence of its complex Young's modulus can be modeled by the Chasset-Thirion law [39, 44, 51, 52]:

$$E(\omega) = E'(\omega) + jE''(\omega) = E_0(1 + (j\omega\tau)^n), \quad (17)$$

where E_0 indicates the static ($\omega = 0$) Young's modulus, τ is the relaxation time, and n is an empirical exponent, which all depend on the sample preparation (i.e., curing procedure, stoichiometric ratio between PDMS and curing agent).

To characterize the static modulus E_0 of the substrate, the dc component of the cantilever's deflection versus the piezo displacement is also recorded, which allows us to construct the force-indentation curve. The static force F is obtained by multiplying the dc component of the deflection by the spring constant k_c of the cantilever. The indentation depth δ is obtained by subtracting the cantilever's deflection from the piezo displacement. The relation between the

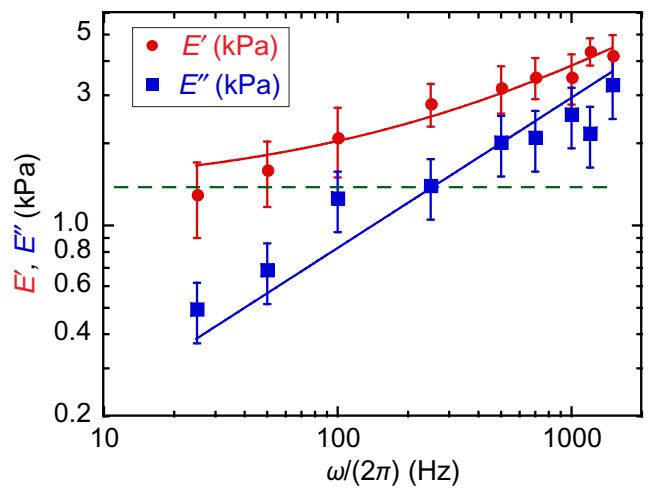


FIG. 4. Storage modulus E' (red) and loss modulus E'' (blue) obtained from the fit in Fig. 3 versus the frequency. The green dashed line represents the value of static modulus $E_0 = 1.3 \pm 0.1$ kPa measured from the independent indentation measurement. The solid lines are fits from Eq. (17), from which we get two free parameters: $n = 0.55 \pm 0.05$ and $\tau = 0.9 \pm 0.1$ ms.

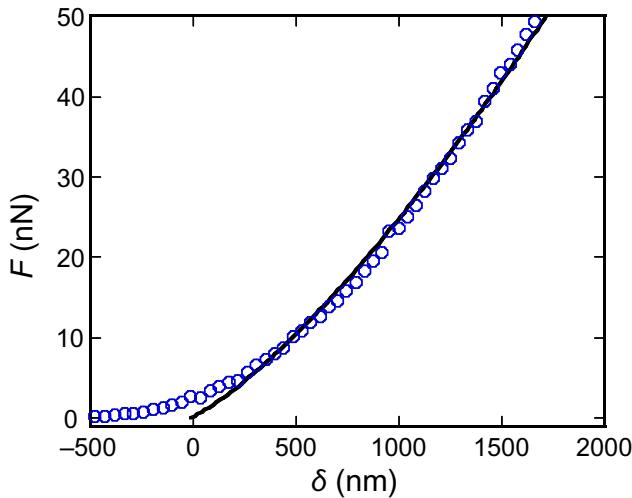


FIG. 5. Measured static force F versus indentation depth δ . The solid line shows the best fit to Eq. (18), from which we obtain the static Young's modulus $E_0 = 1.3 \pm 0.1$ kPa.

force and indentation is given by the well-known Hertz model:

$$F = \frac{16}{9} E_0 R^{1/2} \delta^{3/2}. \quad (18)$$

Figure 5 shows the measured force F as a function of the indentation depth δ . The solid line represents the best fit from Eq. (18). From the fit, we obtain the static Young's modulus $E_0 = 1.3 \pm 0.1$ kPa as a single free parameter. In order to neglect the hydrodynamic force due to the velocity of the substrate approaching the probe, we only perform the fitting at large indentation depths.

With the obtained value of the static modulus E_0 , the extracted storage and loss moduli are fitted by Eq. (17) where the relaxation time τ and the exponent n are taken as fitting parameters. In Fig. 4, the solid lines are the best fits, from which we obtain $n = 0.55 \pm 0.05$ and $\tau = 0.9 \pm 0.1$ ms. The empirical exponent n typically takes values between 1/2 and 2/3 [53].

As a final remark, we recall that the PDMS layer is observed to swell and to increase in thickness by a factor of approximately 5 when immersed in dodecane. Therefore, the PDMS layer contains a large amount of dodecane molecules that may flow across the pores of the network upon compression. Remarkably, the gel appears here to behave mechanically as a viscoelastic material following the Chasset-Thirion law, with no clear evidence of poroelastic response.

V. CONCLUSION

In conclusion, we have studied the viscoelastic response of cross-linked PDMS substrates at several frequencies, using a contactless AFM method. Specifically, a spherical colloidal probe attached to the AFM cantilever is vibrated

in a liquid environment close and normally to a PDMS gel, in order to generate a nanoscale lubrication flow within the gap. This viscous flow generates a hydrodynamic pressure that deforms the soft substrate, leading to an EHD coupling. Based on soft-lubrication theory, we develop a model to calculate the resulting mechanical force exerted on the colloidal probe, including the viscoelasticity of the substrate. Using this model, the storage and loss elastic moduli of the PDMS gel are measured as functions of the frequency. The frequency dependencies of the storage and loss moduli are found to be in good agreement with the Chasset-Thirion law for viscoelastic solids. Altogether, this work demonstrates the robustness of broadband contactless AFM rheological methods. Such methods might be of interest for the gentle and precise frequency-dependent investigation of the mechanical behavior of thin, soft, fragile, immersed and/or alive systems, such as polymer coatings, bubbles, biological membranes, etc. These methods can be used to simultaneously image samples and measure their viscoelastic properties with a lateral spatial resolution limited only by the hydrodynamic radius.

ACKNOWLEDGMENTS

The authors thank Elisabeth Charlaix, Hamid Kellay, Dongshi Guan, Yacine Amarouchene, and Nicolas Fares for fruitful discussions. They acknowledge financial support from the Agence Nationale de la Recherche (Grants No. ANR-19-CE30-0012, No. ANR-18-CE06-0009-03, No. ANR-21-ERCC-0010-01 *EMetBrown*, No. ANR-21-CE06-0039 *Fricolas* and No. ANR-21-CE06-0029 *Softer*) and the China Scholarship Council. They also thank the Soft Matter Collaborative Research Unit, Frontier Research Center for Advanced Material and Life Science, Faculty of Advanced Life Science at Hokkaido University, Sapporo, Japan.

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